



# PDS37-125

3/8-16 X 1-1/4 Pitch Diameter, Stainless Steel, Weld Stud

## Properties

Ferrule: 37P  
 Welding Process: Arc  
 Material: Stainless 302/304 (302/302HQ/304/18-8 Stainless Steel)  
 Annealed: No  
 Nickel Plate: No  
 Weight: 30 Lbs/1,000

## General Attributes

Thread Size: 3/8-16  
 Pitch: 16  
 Weld Base Diameter: 0.33  
 Overall Length: 1-1/4  
 Overall Length (Decimal): 1.25  
 Base Height: 0.385  
 Flash Diameter: 0.49  
 Flash Height: 0.14  
 Flash Clearance: 0.562  
 After Weld Length Reduction (approx.): 0.125

## Mechanical Properties

### Fastener Dimensional Properties

Nominal Diameter (in): 0.375  
 Threads per Inch: 16.00  
 Calculated Pitch Diameter (in): 0.334  
 Calculated Minor Diameter (in): 0.294  
 Effective Weldbase Diameter: N/A  
 (SAF) Calculated Stress Area Full Body: N/A  
 (SAP) Calculated Stress Area Pitch: N/A  
 (SAT) Calculated Stress Area Threads (in): 0.0775 (a)

### Material Properties

(UTM) Ultimate Tensile Strength Min (PSI)/(MPa): 70,000 (b)  
 (YTM) Yield Tensile Strength Min (PSI)/(MPa): 35,000 (b)  
 (PTM) Proof Tensile Strength Min(PSI)/(MPa) 90% YTM: 31,500 (c)  
 (YSM) Yield Shearing Stress Min (PSI)/(MPa): 20,195 (c)

### Fastener Body Mechanical Properties

#### Tensile

Full Diameter Yield Tensile Strength Min (YTM\*SAF): N/A  
 Pitch Diameter Yield Tensile Strength Min (YTM\*SAP): N/A

### Thread Mechanical Properties

#### Tensile

Ultimate Tensile Strength Min (UTM\*SAT): 5,424  
 Yield Tensile Strength Min (YTM\*SAT): 2,712  
 Proof Load Tensile Strength (PTM\*SAT): 2,441  
 Preload Reused Tensile (75% PL): 1,831 (d)  
 Preload Permanent (90% PL): 2,197 (e)

#### Torque

Friction Coefficient: 0.2 (f)  
 Torque Reused (FC\*PR\*ND): 137.3  
 Torque Permanent (FC\*PP\*ND): 164.8  
 Torque Ultimate (FC\*UTS\*ND): 406.8 (g)

#### Shear

Yield Shear Min (YSM\*SAT): 1564.9 (h)

#### Shear

Full Diameter Yield Shear Min (YSM\*SAF): N/A  
 Pitch Diameter Yield Shear Min (YSM\*SAP): N/A

- a) Stress area for the threads is calculated on an area approximately half way between the root diameter and pitch diameter. This more closely reflects actual results versus using root diameter for stress area calculations.
- b) Data provided by our suppliers.
- c) Proof Tensile is estimated to be 90% of Yield Tensile when Proof Tensile data is not readily available.
- d) Preload for reuse of threaded fastener is 75% of Proof Load. Fastener may be rebolted multiple times with no degradation.
- e) Preload for permanent installation of fastener is 90% of Proof Load. Fastener is permanently stretched and will not achieve the same clamping force on reuse.
- f) Friction Coefficient will vary greatly depending on bolting conditions including lubrication. 0.2 is standard when bolt condition is plain finish or unknown.
- g) Ultimate Torque is not useful in designing a bolted joint. Preloaded torque is more practical. This data is provided as a reference only.
- h) Using the distortion-energy theory; Maximum Shear Stress equals .577 times the Maximum Tensile.

### CAUTION:

Fasteners should not be used at their tensile or shear limits. A safety factor must be applied to engineering calculations. The particular safety factor will vary depending on the application.

**Customers should always evaluate fasteners for suitability of their own applications.**