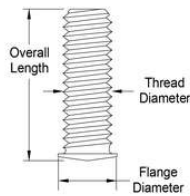
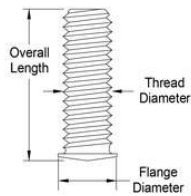


Image Industries Data Sheet



APA11-50

#4-40 X 1/2 Advanced Process, Aluminum, Weld Stud

Properties

Ferrule: N/A
Welding Process: Gas
Material: Aluminum (Aluminum 5356)
Weight: 0.45 Lbs/1,000

General Attributes

Thread Size: #4-40
Thread Pitch: 40
Body Diameter: 0.11
Flange Diameter: 0.142
Overall Length: 1/2
Overall Length (Decimal): 0.5
After Weld Length Reduction (approximately): 0.06

Mechanical Properties

Fastener Dimensional Properties

Nominal Diameter (in): 0.112
Threads per Inch: 40.00
Calculated Pitch Diameter (in): 0.096
Calculated Minor Diameter (in): 0.08
Effective Weldbase Diameter: N/A
(SAF) Calculated Stress Area Full Body: N/A
(SAP) Calculated Stress Area Pitch: N/A
(SAT) Calculated Stress Area Threads (in): 0.006 (a)

Material Properties

(UTM) Ultimate Tensile Strength Min (PSI)/(MPa): 35,000 (b)
(YTM) Yield Tensile Strength Min (PSI)/(MPa): 19,000 (b)
(PTM) Proof Tensile Strength Min(PSI)/(MPa) 90% YTM: 17,100 (c)
(YSM) Yield Shearing Stress Min (PSI)/(MPa): 10,963 (c)

Fastener Body Mechanical Properties

Tensile

Full Diameter Yield Tensile Strength Min (YTM*SAF): N/A
Pitch Diameter Yield Tensile Strength Min (YTM*SAP): N/A

Thread Mechanical Properties

Tensile

Ultimate Tensile Strength Min (UTM*SAT): 211
Yield Tensile Strength Min (YTM*SAT): 115
Proof Load Tensile Strength (PTM*SAT): 103
Preload Reused Tensile (75% PL): 77 (d)
Preload Permanent (90% PL): 93 (e)

Torque

Friction Coefficient: 0.2 (f)
Torque Reused (FC*PR*ND): 1.7
Torque Permanent (FC*PP*ND): 2.1
Torque Ultimate (FC*UTS*ND): 4.7 (g)

Shear

Yield Shear Min (YSM*SAT): 66.1 (h)

Shear

Full Diameter Yield Shear Min (YSM*SAF): N/A
Pitch Diameter Yield Shear Min (YSM*SAP): N/A

a) Stress area for the threads is calculated on an area approximately half way between the root diameter and pitch diameter.

This more closely reflects actual results versus using root diameter for stress area calculations.

b) Data provided by our suppliers.

c) Proof Tensile is estimated to be 90% of Yield Tensile when Proof Tensile data is not readily available.

d) Preload for reuse of threaded fastener is 75% of Proof Load. Fastener may be rebolted multiple times with no degradation.

e) Preload for permanent installation of fastener is 90% of Proof Load. Fastener is permanently stretched and will not achieve the same clamping force on reuse.

f) Friction Coefficient will vary greatly depending on bolting conditions including lubrication. 0.2 is standard when bolt condition is plain finish or unknown.

g) Ultimate Torque is not useful in designing a bolted joint. Preloaded torque is more practical. This data is provided as a reference only.

h) Using the distortion-energy theory; Maximum Shear Stress equals .577 times the Maximum Tensile.

CAUTION:

Fasteners should not be used at their tensile or shear limits. A safety factor must be applied to engineering calculations. The particular safety factor will vary depending on the application.

Customers should always evaluate fasteners for suitability of their own applications.

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